

Notes

- ◆ More optional reading on web for collision detection

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Triangles

- ◆ Given x_1, x_2, x_3 the plane normal is

$$n = \frac{(x_2 - x_1) \times (x_3 - x_1)}{|(x_2 - x_1) \times (x_3 - x_1)|}$$

- ◆ Interference with a closed mesh
 - Cast a ray to infinity, parity of number of intersections gives inside/outside
- ◆ So intersection is more fundamental
 - The same problem as in ray-tracing

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Triangle intersection

- ◆ The best approach: reduce to simple predicates
 - Spend the effort making them exact, accurate, or at least consistent
 - Then it's just some logic on top
 - Common idea in computational geometry
- ◆ In this case, predicate is sign of signed volume (is a tetrahedra inside-out?)

$$\text{orient}(x_0, x_1, x_2, x_3) = \text{sign} \det \begin{pmatrix} x_1 - x_0 & y_1 - y_0 & z_1 - z_0 \\ x_2 - x_0 & y_2 - y_0 & z_2 - z_0 \\ x_3 - x_0 & y_3 - y_0 & z_3 - z_0 \end{pmatrix}$$

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Using orient()

- ◆ Line-triangle
 - If line includes x_4 and x_5 then intersection if $\text{orient}(1,2,4,5) = \text{orient}(2,3,4,5) = \text{orient}(3,1,4,5)$
 - I.e. does the line pass to the left (right) of each directed triangle edge?
 - If normalized, the values of the determinants give the **barycentric coordinates** of plane intersection point
- ◆ Segment-triangle
 - Before checking line as above, also check if $\text{orient}(1,2,3,4) \neq \text{orient}(1,2,3,5)$
 - I.e. are the two endpoints on different sides of the triangle?

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Other Standard Approach

- ◆ Find where line intersects plane of triangle
- ◆ Check if it's on the segment
- ◆ Find if that point is inside the triangle
 - Use barycentric coordinates
- ◆ Slightly slower, but worse: less robust
 - round-off error in intermediate result: the intersection point
 - What happens for a triangle mesh?
- ◆ Note the predicate approach, even with floating-point, can handle meshes well
 - Consistent evaluation of predicates for neighbouring triangles

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Distance to Triangle

- ◆ If surface is open, define interference in terms of distance to mesh
- ◆ Typical approach: find closest point on triangle, then distance to that point
 - Direction to closest point also parallel to natural normal
- ◆ First step: barycentric coordinates
 - Normalized signed volume determinants equivalent to solving least squares problem of closest point in plane
- ◆ If coordinates all in $[0,1]$ we're done
- ◆ Otherwise negative coords identify possible closest edges
- ◆ Find closest points on edges

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Testing Against Meshes

- ◆ Can check every triangle if only a few, but too slow usually
- ◆ Use an acceleration structure:
 - Spatial decomposition: background grid, hash grid, octree, kd-tree, BSP-tree, ...
 - Bounding volume hierarchy: axis-aligned boxes, spheres, oriented boxes, ...

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Moving Triangles

- ◆ Collision detection: find a time at which particle lies inside triangle
- ◆ Need a model for what triangle looks like at intermediate times
 - Simplest: vertices move with constant velocity, triangle always just connects them up
- ◆ Solve for intermediate time when four points are coplanar (determinant is zero)
 - Gives a cubic equation to solve
- ◆ Then check barycentric coordinates at that time
 - See e.g. X. Provot, "Collision and self-collision handling in cloth model dedicated to design garment", Graphics Interface'97

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For Later...

- ◆ We now can do all the basic particle vs. object tests for repulsions and collisions
- ◆ Once we get into simulating solid objects, we'll need to do object vs. object instead of just particle vs. object
- ◆ Core ideas remain the same

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Elasticity

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Elastic objects

- ◆ Simplest model: masses and springs
- ◆ Split up object into regions
- ◆ Integrate density in each region to get mass (if things are uniform enough, perhaps equal mass)
- ◆ Connect up neighbouring regions with springs
 - Careful: need chordal graph
- ◆ Now it's just a particle system
 - When you move a node, neighbours pulled along with it, etc.

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Masses and springs

- ◆ But: how strong should the springs be? Is this good in general?
 - [anisotropic examples]
- ◆ General rule: we don't want to see the mesh in the output
 - Avoid "grid artifacts"
 - We of course will have numerical error, but let's avoid obvious patterns in the error

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1D masses and springs

- ◆ Look at a homogeneous elastic rod, length 1, linear density ρ
- ◆ Parameterize by p ($x(p)=p$ in rest state)
- ◆ Split up into intervals/springs
 - $0 = p_0 < p_1 < \dots < p_n = 1$
 - Mass $m_i = \rho(p_{i+1} - p_i)/2$ (+ special cases for ends)
 - Spring $i+1/2$ has rest length

and force

$$L_{i+1/2} = p_{i+1} - p_i$$

$$f_{i+1/2} = k_{i+1/2} \frac{x_{i+1} - x_i - L_{i+1/2}}{L_{i+1/2}}$$

Figuring out spring constants

- ◆ So net force on i is

$$F_i = k_{i+1/2} \frac{x_{i+1} - x_i - L_{i+1/2}}{L_{i+1/2}} - k_{i-1/2} \frac{x_i - x_{i-1} - L_{i-1/2}}{L_{i-1/2}}$$

$$= k_{i+1/2} \left(\frac{x_{i+1} - x_i}{p_{i+1} - p_i} - 1 \right) - k_{i-1/2} \left(\frac{x_i - x_{i-1}}{p_i - p_{i-1}} - 1 \right)$$

- ◆ We want mesh-independent response (roughly), e.g. for static equilibrium
 - Rod stretched the same everywhere: $x_i = \alpha p_i$
 - Then net force on each node should be zero (add in constraint force at ends...)

Young's modulus

- ◆ So each spring should have the same k
 - Note we divided by the rest length
 - Some people don't, so they have to make their constant scale with rest length
- ◆ The constant k is a material property (doesn't depend on our discretization) called the Young's modulus
 - Often written as E
- ◆ The one-dimensional Young's modulus is simply force per percentage deformation

The continuum limit

- ◆ Imagine Δp (or Δx) going to zero
 - Eventually can represent any kind of deformation
 - [note force and mass go to zero too]

$$\ddot{x}(p) = \frac{1}{\rho} \frac{\partial}{\partial p} \left(E(p) \left(\frac{\partial}{\partial a} x(p) - 1 \right) \right)$$

- If density and Young's modulus constant,

$$\frac{\partial^2 x}{\partial t^2} = \frac{E}{\rho} \frac{\partial^2 x}{\partial p^2}$$

Sound waves

- ◆ Try solution $x(p,t) = x_0(p-ct)$
- ◆ And $x(p,t) = x_0(p+ct)$
- ◆ So speed of "sound" in rod is $\sqrt{\frac{E}{\rho}}$
- ◆ Courant-Friedrichs-Levy (CFL) condition:
 - Numerical methods only will work if information transmitted numerically at least as fast as in reality (here: the speed of sound)
 - Usually the same as stability limit for good explicit methods [what are the eigenvalues here]
 - Implicit methods transmit information infinitely fast