

Notes

- ◆ I am back, but still catching up
- ◆ Assignment 2 is due today (or next time I'm in the dept following today)
- ◆ Final project proposals:
 - I haven't sorted through my email, but make sure you send me something now (even quite vague)
 - Let's make sure everyone has their project started this weekend or early next week

Multi-Dimensional Plasticity

- ◆ Simplest model: total strain is sum of elastic and plastic parts: $\epsilon = \epsilon_e + \epsilon_p$
- ∪ Stress only depends on elastic part (so rest state includes plastic strain):
 $\sigma = \sigma(\epsilon_e)$
- ∪ If σ is too big, we yield, and transfer some of ϵ_e into ϵ_p so that σ is acceptably small

Multi-Dimensional Yield criteria

- ◆ Lots of complicated stuff happens when materials yield
 - Metals: dislocations moving around
 - Polymers: molecules sliding against each other
 - Etc.
- ◆ Difficult to characterize exactly when plasticity (yielding) starts
 - Work hardening etc. mean it changes all the time too
- ◆ Approximations needed
 - Big two: Tresca and Von Mises

Yielding

- ◆ First note that shear stress is the important quantity
 - Materials (almost) never can permanently change their volume
 - Plasticity should ignore volume-changing stress
- ◆ So make sure that if we add kl to σ it doesn't change yield condition

Tresca yield criterion

- ◆ This is the simplest description:
 - Change basis to diagonalize σ
 - Look at normal stresses (i.e. the eigenvalues of σ)
 - No yield if $\sigma_{\max} - \sigma_{\min} \leq \sigma_Y$
- ∪ Tends to be conservative (rarely predicts yielding when it shouldn't happen)
- ∪ But, not so accurate for some stress states
 - Doesn't depend on middle normal stress at all
- ∪ Big problem (mathematically): not smooth

Von Mises yield criterion

- ◆ If the stress has been diagonalized:

$$\frac{1}{\sqrt{2}} \sqrt{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2} \leq \sigma_Y$$
- ◆ More generally:

$$\sqrt{\frac{3}{2}} \sqrt{\|\sigma\|_F^2 - \frac{1}{3} \text{Tr}(\sigma)^2} \leq \sigma_Y$$
- ◆ This is the same thing as the Frobenius norm of the deviatoric part of stress
 - i.e. after subtracting off volume-changing part:

$$\sqrt{\frac{3}{2}} \left\| \sigma - \frac{1}{3} \text{Tr}(\sigma) I \right\|_F \leq \sigma_Y$$

Linear elasticity shortcut

- ◆ For linear (and isotropic) elasticity, apart from the volume-changing part which we cancel off, stress is just a scalar multiple of strain
 - (ignoring damping)
- ◆ So can evaluate von Mises with elastic strain tensor too (and an appropriately scaled yield strain)

Perfect plastic flow

- ◆ Once yield condition says so, need to start changing plastic strain
- ◆ The magnitude of the change of plastic strain should be such that we stay on the yield surface
 - i.e. maintain $f(\sigma)=0$
(where $f(\sigma) \leq 0$ is, say, the von Mises condition)
- ◆ The direction that plastic strain changes isn't as straightforward
- ◆ "Associative" plasticity:

$$\dot{\epsilon}_p = \gamma \frac{\partial f}{\partial \sigma}$$

Algorithm

- ◆ After a time step, check von Mises criterion:

$$\text{is } f(\sigma) = \sqrt{\frac{3}{2}} \|\text{dev}(\sigma)\|_F - \sigma_Y > 0 \text{ ?}$$

- ◆ If so, need to update plastic strain:

$$\begin{aligned} \varepsilon_p^{\text{new}} &= \varepsilon_p + \gamma \frac{\partial f}{\partial \sigma} \\ &= \varepsilon_p + \gamma \sqrt{\frac{3}{2}} \frac{\text{dev}(\sigma)}{\|\text{dev}(\sigma)\|_F} \end{aligned}$$

- with γ chosen so that $f(\sigma^{\text{new}}) = 0$
(easy for linear elasticity)

Sand (Granular Materials)

- ◆ Things get a little more complicated for sand, soil, powders, etc.
- ◆ Yielding actually involves friction, and thus is pressure (the trace of stress) dependent
- ◆ Flow rule can't be associated
- ◆ See Zhu and Bridson, SIGGRAPH'05 for quick-and-dirty hacks... :-)

Multi-Dimensional Fracture

- ◆ Smooth stress to avoid artifacts (average with neighbouring elements)
- ◆ Look at largest eigenvalue of stress in each element
- ◆ If larger than threshold, introduce crack perpendicular to eigenvector
- ◆ Big question: what to do with the mesh?
 - Simplest: just separate along closest mesh face
 - Or split elements up: O'Brien and Hodgins SIGGRAPH'99
 - Or model crack path with embedded geometry: Molino et al. SIGGRAPH'04

Fluids

Fluid mechanics

- ◆ We already figured out the equations of motion for continuum mechanics $\rho \ddot{x} = \nabla \cdot \sigma + \rho g$

- ◆ Just need a constitutive model

$$\sigma = \sigma(x, t, \varepsilon, \dot{\varepsilon})$$

- ◆ We'll look at the constitutive model for "Newtonian" fluids next
 - Remarkably good model for water, air, and many other simple fluids
 - Only starts to break down in extreme situations, or more complex fluids (e.g. viscoelastic substances)

Inviscid Euler model

- ◆ Inviscid=no viscosity
- ◆ Great model for most situations
 - Numerical methods usually end up with viscosity-like error terms anyways...
- ◆ Constitutive law is very simple: $\sigma_{ij} = -p\delta_{ij}$
 - New scalar unknown: pressure p
 - Barotropic flows: p is just a function of density (e.g. perfect gas law $p=k(\rho-\rho_0)+p_0$ perhaps)
 - For more complex flows need heavy-duty thermodynamics: an equation of state for pressure, equation for evolution of internal energy (heat), ...

Lagrangian viewpoint

- ◆ We've been working with Lagrangian methods so far
 - Identify chunks of material, track their motion in time, differentiate world-space position or velocity w.r.t. material coordinates to get forces
 - In particular, use a mesh connecting particles to approximate derivatives (with FVM or FEM)
- ◆ Bad idea for most fluids
 - [vortices, turbulence]
 - At least with a fixed mesh...

Eulerian viewpoint

- ◆ Take a fixed grid in world space, track how velocity changes at a point
- ◆ Even for the craziest of flows, our grid is always nice
- ◆ (Usually) forget about object space and where a chunk of material originally came from
 - Irrelevant for extreme inelasticity
 - Just keep track of velocity, density, and whatever else is needed

Conservation laws

- ◆ Identify any fixed volume of space
- ◆ Integrate some conserved quantity in it (e.g. mass, momentum, energy, ...)
- ◆ Integral changes in time only according to how fast it is being transferred from/to surrounding space

$$\frac{\partial}{\partial t} \int_{\Omega} q = - \int_{\partial\Omega} f(q) \cdot n$$

- Called the flux
- [divergence form] $q_t + \nabla \cdot f = 0$

Conservation of Mass

- ◆ Also called the continuity equation (makes sure matter is continuous)
- ◆ Let's look at the total mass of a volume (integral of density)
- ◆ Mass can only be transferred by moving it: flux must be ρu

$$\frac{\partial}{\partial t} \int_{\Omega} \rho = - \int_{\partial\Omega} \rho u \cdot n$$

$$\rho_t + \nabla \cdot (\rho u) = 0$$

Material derivative

- ◆ A lot of physics just naturally happens in the Lagrangian viewpoint
 - E.g. the acceleration of a material point results from the sum of forces on it
 - How do we relate that to rate of change of velocity measured at a fixed point in space?
 - Can't directly: need to get at Lagrangian stuff somehow
- ◆ The material derivative of a property q of the material (i.e. a quantity that gets carried along with the fluid) is $\frac{Dq}{Dt}$

Finding the material derivative

- ◆ Using object-space coordinates p and map $x=X(p)$ to world-space, then material derivative is just

$$\begin{aligned} \frac{D}{Dt} q(t, x) &= \frac{d}{dt} q(t, X(t, p)) \\ &= \frac{\partial q}{\partial t} + \nabla q \cdot \frac{\partial x}{\partial t} \\ &= q_t + u \cdot \nabla q \end{aligned}$$

- ◆ Notation: u is velocity (in fluids, usually use u but occasionally v or V , and components of the velocity vector are sometimes u, v, w)

Compressible Flow

- ◆ In general, density changes as fluid compresses or expands
- ◆ When is this important?
 - Sound waves (and/or high speed flow where motion is getting close to speed of sound - Mach numbers above 0.3?)
 - Shock waves
- ◆ Often not important scientifically, almost never visually significant
 - Though the **effect** of e.g. a blast wave is visible! But the shock dynamics usually can be hugely simplified for graphics

Incompressible flow

- ◆ So we'll just look at incompressible flow, where density of a chunk of fluid never changes
 - Note: fluid density may not be constant throughout space - different fluids mixed together...
- ◆ That is, $D\rho/Dt=0$

Simplifying

- ◆ Incompressibility: $\frac{D\rho}{Dt} = \rho_t + u \cdot \nabla \rho = 0$
- ◆ Conservation of mass: $\rho_t + \nabla \cdot (\rho u) = 0$
 $\rho_t + \nabla \rho \cdot u + \rho \nabla \cdot u = 0$
- ◆ Subtract the two equations, divide by ρ :

$$\nabla \cdot u = 0$$

- ◆ Incompressible == divergence-free velocity
 - Even if density isn't uniform!

Conservation of momentum

- ◆ Short cut: in $\rho \ddot{x} = \nabla \cdot \sigma + \rho g$

use material derivative:

$$\rho \frac{Du}{Dt} = \nabla \cdot \sigma + \rho g$$

$$\rho(u_t + u \cdot \nabla u) = \nabla \cdot \sigma + \rho g$$

- ◆ Or go by conservation law, with the flux due to transport of momentum and due to stress:
 - Equivalent, using conservation of mass

$$(\rho u)_t + \nabla \cdot (u \rho u - \sigma) = \rho g$$

Inviscid momentum equation

- ◆ Plug in simplest constitutive law ($\sigma = -p\delta$) from before to get

$$\rho(u_t + u \cdot \nabla u) = -\nabla p + \rho g$$

$$u_t + u \cdot \nabla u + \frac{1}{\rho} \nabla p = g$$

- Together with conservation of mass: the Euler equations

Incompressible inviscid flow

- ◆ So the equations are:
$$u_t + u \cdot \nabla u + \frac{1}{\rho} \nabla p = g$$
$$\nabla \cdot u = 0$$
- ◆ 4 equations, 4 unknowns (u, p)
- ◆ Pressure p is just whatever it takes to make velocity divergence-free
 - Actually a “Lagrange multiplier” for enforcing the incompressibility constraint

Pressure solve

- ◆ To see what pressure is, take divergence of momentum equation

$$\nabla \cdot (u_t + u \cdot \nabla u + \frac{1}{\rho} \nabla p - g) = 0$$

$$\nabla \cdot (\frac{1}{\rho} \nabla p) = -\nabla \cdot (u_t + u \cdot \nabla u - g)$$

- ◆ For constant density, just get Laplacian (and this is Poisson’s equation)
- ◆ Important numerical methods use this approach to find pressure

Projection

- ◆ Note that $\nabla \cdot u_t = 0$ so in fact

$$\nabla \cdot \frac{1}{\rho} \nabla p = -\nabla \cdot (u \cdot \nabla u - g)$$

- ∪ After we add $\nabla p / \rho$ to $u \cdot \nabla u$, divergence must be zero
- ∪ So if we tried to solve for additional pressure, we get zero
- ∪ Pressure solve is linear too
- ∪ Thus what we’re really doing is a **projection** of $u \cdot \nabla u - g$ onto the subspace of divergence-free functions:
 $u_t + P(u \cdot \nabla u - g) = 0$